Lower limit on the achievable temperature in resonator-based sideband cooling

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A resonator with eigenfrequency $\omega_r$ can be effectively used as a cooler for another linear oscillator with a much smaller frequency $\omega_0 \ll \omega_r$. A huge cooling effect, which could be used to cool a mechanical oscillator below the energy of quantum fluctuations, has been predicted by several authors. However, here we show that there is a lower limit $T^*$ on the achievable temperature, given by $T^* = \frac{T_0^* \omega_m}{\omega_r}$, that was not considered in previous work and can be higher than the quantum limit in realistic experimental realizations. We also point out that the decay rate of the resonator, which previous studies stress should be small, must be larger than the decay rate of the cooled oscillator for effective cooling.

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I. INTRODUCTION

Recently, a tremendous experimental effort has been devoted to the task of cooling mechanical oscillators below the energy of quantum fluctuations. In spite of many experimental improvements, the quantum limit has not been achieved.1–5 Several papers that propose cooling mechanisms using electromagnetic (radio frequency, microwave or light) resonators6–9 or other cooling mechanisms10–15 to fulfill this task have appeared recently. These papers predict an enormous cooling effect. However, they do not explicitly state that there is a lower limit on the achievable temperature, associated with the ratio between the frequencies of the coolant and cooled oscillators, which cannot be overcome and can play an important role for realistic experimental realizations. Moreover, some formulas that appear in the literature can give temperatures below this limit, which will be described in more detail below. This lower temperature limit can be important for the most feasible designs using radio frequency or microwave resonators.

II. SEMICLASSICAL APPROACH

For the sake of simplicity, we will consider a RLC tank circuit (the results can be applied to any electromagnetic resonator, such as a transmission-line resonator, cavity, Fabry-Pérot resonator, etc.). A mechanical oscillator is coupled to the capacitor such that the capacitance depends parametrically on the displacement of the oscillator. Such a system was thoroughly analyzed in Ref. 23, and we only briefly introduce the equations of motion here. If the mechanical oscillator is a part of one of the capacitor electrodes, the capacitance $C(x) = C_0(1-x/d)$ depends on the displacement $x$ of the oscillator from the equilibrium position, where $C_0 = \epsilon S/d_0$ is the capacitance at $x=0$, and $d$ is the renormalized distance between the electrodes $d = d_0/\kappa$. Here $\kappa$ is the coupling constant between the mechanical oscillator and the RLC circuit, and it can be expressed as the ratio between the mechanical oscillator capacitance $C_m$, which depends on the oscillator displacement, and the total capacitance $C_0$ (we consider the case $C_m \ll C_0$), i.e., $\kappa = C_m/C_0$. If the RLC tank circuit is pumped by a microwave source $V_f = V_{0f} \cos \omega_0 t$, the voltage between the capacitor’s electrodes is $V_0 = \omega_0 V_{0f}/\Gamma$, and the Coulomb energy of the capacitor depends on its capacitance, which in turn, depends on the oscillator displacement. Thus, the electromagnetic resonator and mechanical oscillator (i.e., cantilever) can be described by a system of differential equations of two coupled damped linear oscillators:

$$\frac{d^2 Q}{dt^2} + \Gamma_r \frac{dQ}{dt} + \omega_r^2 Q = \frac{V_p(t) + V_f(t)}{L},$$  

$$\frac{d^2 x}{dt^2} + \Gamma_m \frac{dx}{dt} + \omega_m^2 x = \frac{F_f(t)}{M} + \frac{Q(t)^2}{2MC_0d},$$

where $\Gamma_{r,m}$ are damping rates, $\omega_{r,m}$ are angular frequencies, $V_f$ is a fluctuating voltage across the capacitor, $F_f$ is a fluctuating force acting on the mechanical oscillator with mass $M$, and $Q(t) = q_p(t) + q_f(t)$ is the total charge on the capacitor. Equation (2) is nonlinear but can be linearized keeping in
mind that we are interested to calculate charge fluctuations $q_z(t)$, which are much smaller than charge oscillations $q_p(t)$ driving by coherent microwave source. It is convenient to express $q_z(t)$ and $V_z(t)$ in terms of quadrature amplitudes

$$q_z(t) = q_z(t) \cos \omega_p t + q_z(t) \sin \omega_p t,$$

$$V_z(t) = V_z(t) \cos \omega_p t + V_z(t) \sin \omega_p t,$$

and rewrite Eqs. (1) and (2) in the dimensionless variables

$$\bar{q}_{c,i} = q_{c,i} \sqrt{C_0 h \omega_r},$$

$$\bar{x} = x / \sqrt{h \omega_p / M \omega_m^2},$$

$$\tau = \omega_p t,$$

$$\bar{\Gamma}_m = \Gamma_m / \omega_m,$$

$$\bar{\omega}_{r,p} = \omega_{r,p} / \omega_m,$$

$$\bar{\Gamma}_r = \Gamma_r / 2 \omega_m,$$

$$\bar{V}_{c,i} = \bar{\omega}_i V_{c,i} \sqrt{C_0 / h \omega_r},$$

$$\bar{F}_i = F_i / \sqrt{M \omega_m^2 h \omega_m},$$

$$\bar{V}_0 = \bar{\omega}_0 V_0 \sqrt{C_0 / 4 M \omega_m^2 \omega_d^2}.$$

Here $T_r$ and $T_m$ are the temperatures of the electromagnetic resonator and mechanical oscillator, respectively. Considering Langevin fluctuating forces caused by quantum noise,24,25

$$V_{c,i}(t) = \sqrt{L \Gamma_r h \omega_r / 2} \coth \left( \frac{h \omega_r}{2k_B T_r} \right) \bar{\xi}_{c,i}(t),$$

$$F_i(t) = \sqrt{M \Gamma_m h \omega_m \coth \left( \frac{h \omega_m}{2k_B T_m} \right)} \bar{\xi}_m(t),$$

and using the slowly-varying-amplitude approximation23 Eqs. (1) and (2) read

$$\frac{d\bar{q}}{d\tau} = -\bar{A} \bar{q}(\tau) + \bar{F}(\tau),$$

where

$$\bar{A} = \begin{pmatrix} \bar{\Gamma}_r & -\bar{\eta} & 0 & 0 \\ -\bar{\eta} & \bar{\Gamma}_r & 0 & -\bar{V}_0 \\ -\bar{V}_0 & 0 & \bar{\Gamma}_m & 1 \\ 0 & 0 & -1 & 0 \end{pmatrix},$$

$$\bar{F}(\tau) = \begin{pmatrix} \sqrt{\bar{\Gamma}_r} \coth \left( \frac{h \omega_r}{2k_B T_r} \right) \bar{\xi}(\tau) \\ \sqrt{\bar{\Gamma}_r} \coth \left( \frac{h \omega_r}{2k_B T_r} \right) \bar{\xi}(\tau) \\ \sqrt{\bar{\Gamma}_m} \coth \left( \frac{h \omega_m}{2k_B T_m} \right) \bar{\xi}_m(\tau) \\ 0 \end{pmatrix}.$$

The mean value of energy of the mechanical oscillator fluctuations is

$$\mathcal{E}_m = \sigma_{\bar{v} \bar{v}} \hbar \omega_m.$$

Now, one can easily calculate the effective temperature of the mechanical oscillator from the definition relation for $T_m'$

$$\sigma_{\bar{v} \bar{v}} = \frac{1}{2} \coth \left( \frac{h \omega_m}{2k_B T_m'} \right).$$

As we will see later, the most appropriate parameters for cooling purposes are $\bar{\eta} = -1$, $2 \bar{\Gamma}_m \bar{\Gamma}_r \ll \bar{V}_0 \ll 1$. In this limit and for $\Gamma_r < \omega_m$, $\Gamma_m \ll \Gamma_r$, the $\sigma_{\bar{v} \bar{v}}$ can be expressed as

$$\sigma_{\bar{v} \bar{v}} = \frac{1}{2} \coth \left( \frac{h \omega_m}{2k_B T_m'} \right) + \frac{\bar{\Gamma}_r \bar{\Gamma}_m}{V_0^2} \coth \left( \frac{h \omega_m}{2k_B T_m'} \right).$$

Thus, the lowest temperature of the mechanical oscillator is limited by the first term if the second term is made negligibly small by sideband cooling. As a matter of fact this term simply shows that even in our semiclassical approach, we cannot “cool” the mechanical oscillator below the zero-point energy, which is consistent with the Heisenberg uncertainty principle. Indeed, it follows from Eqs. (7) and (8) that the energy saved in the mechanical oscillator is...
FIG. 1. The cooling factor $T_m^*/T_m$ as a function of the normalized pumping amplitude $\tilde{V}_0$ of the noiseless microwave source for $\omega_r=10^3$, $\Gamma_m=10^{-5}$, $\Gamma_r=10^{-1}$, and $k_BT_m=k_BT_r=\hbar\omega_{r,m}$ calculated numerically (circles) and from Eqs. (8) and (9) (solid line).

In this limit, the effective temperature $T_m^*$ of the mechanical oscillator takes the simple form

$$T_m^* = \frac{\omega_m}{\omega_r} T_r.$$

The cooling factor $T_m^*/T_m$ as a function of the normalized pumping amplitude $\tilde{V}_0$ is shown in Fig. 1. Even though this result was derived within semiclassical physics, the same limit can be obtained using the quantum approach, as we shall show below.

Here we should emphasize that the temperature of the resonator $T_r$ is usually much higher than the ambient temperature if the resonator is heavily pumped by the microwave source. This is caused by the phase noise of the microwave source, which is directly proportional to the output power. Microwave sources are characterized by the single sideband noise spectral density.

$$L(\Delta \omega) = 10 \log \left( \frac{S_V}{U_{\text{rms}}^2} \right),$$

where $U_{\text{rms}}^2$ is the mean square voltage of the microwave source, and $S_V$ is the spectral density of the voltage noise. The effective temperature $T_r$ of the pumped resonator can be calculated as

$$T_r = T_{r0} + \frac{\omega_r}{\Gamma_r} \frac{U_{\text{rms}}^2}{2k_BZ_r} 10^{\frac{L(\Delta \omega)}{10}},$$

where $T_{r0}$ is the temperature of the resonator without pumping and $Z_r=\sqrt{L/C}$ is the characteristic impedance of the resonator. Now, both terms in Eq. (9) depend on the pumping power. The first one increases with pumping power while the second one decreases. Since the highest cooling power is expected for $\tilde{V}_0 \ll 1$, the effective temperature of the mechanical oscillator is higher than

$$T_{m0} \approx \frac{U_{\text{rms}}^2}{2k_BZ_r} 10^{\frac{L(\Delta \omega)}{10}}.$$

Thus, for microwave resonators the first term in Eq. (9) becomes important, especially for the cooling of mechanical oscillators with high resonant frequencies approaching the GHz range. The minimal temperature $T_{m0}$ is directly proportional to the pumping power in the limit $\tilde{V}_0 \ll 1$, which is the relevant limit in order to determine the lowest achievable temperature of the mechanical oscillator cooled by sideband cooling. For present state-of-the-art microwave generators [$L(21 \text{ MHz})=-160 \text{ dBc/Hz}$], the effective temperature of the mechanical oscillator $T_{m0}$ as a function of pumping voltage is shown in Fig. 2. The parameters were chosen according to recently achieved values (see Refs. 27 and 28) as follows: the base temperature, characteristic impedance, and angular frequency of the resonator are given by $T_{r0}=50 \text{ mK}$, $Z_r=50 \Omega$, and $\omega_r=2\pi\times21 \text{ GHz}$, respectively, and the angular frequency of the mechanical oscillator $\omega_m=2\pi\times21 \text{ MHz}$. The quantum regime of the mechanical oscillator can be achieved if the voltage of the microwave source is below 0.1 mV. However, the cooling of the mechanical oscillator by simple coupling to the microwave resonator requires a higher microwave voltage. Therefore, the coupling should be designed to be as strong as possible in order to achieve the quantum regime. For example, for a mechanical oscillator with resonance frequency smaller than 1 MHz, one cannot achieve the quantum limit with realistic microwave sources if the coupling is small. Namely, the cooling of a mechanical oscillator with angular frequency $\omega_m=2\pi\times300 \text{ kHz}$ to the quantum regime, as proposed in
Ref. 7, would require a voltage $U_{\text{mw}} \approx 0.5$ mV. However, the best commercially available microwave sources with frequency $\sim 1$ GHz achieve $L(300$ kHz$) = -150$ dBc/Hz only, and therefore the thermal energy of the mechanical oscillator would be much higher than its zero-point energy for such a microwave voltage. Moreover, even for an ideal microwave source the quantum limit cannot be achieved because of the lower limit for sideband cooling (dashed line in Fig. 2). Superconducting qubits,10,12 which have sizes similar to those of mechanical nano-oscillators, can be a better option than microwave resonators.

### III. Quantum Approach

In order to achieve the quantum regime of the mechanical oscillator, the temperature should be lower than the energy of quantum fluctuations, which, together with Eq. (11), imply the inequality

$$a_m T_m < T < \frac{\hbar a_m}{2k_B}.$$  

Thus the microwave resonator should be in the quantum regime as well, and the classical description is no longer valid. Therefore, we now turn to the analysis of this problem using the quantum description when the resonator’s frequency is higher than its temperature and the resonator is in its ground state with high probability. In this case the cooling limit can be derived in a transparent manner using a thermodynamic approach. Another advantage of this approach is that a large part of the analysis (in particular, the derivation of the lowest achievable temperature) is also valid for nonlinear coolers, including the case where the resonator is substituted by a two-level system (qubit) as suggested in Ref. 10. The thermodynamic approach is also valid regardless of the specific form of the coupling and driving terms in the Hamiltonian, up to some mild requirements that will be explained below.

The Hamiltonian that we shall use in our analysis is given by

$$\hat{H} = a_r^\dagger a_r + \omega_m a_m^\dagger a_m + \hat{H}_{\text{coupling}} + \hat{H}_{\text{drive}},$$  

where $a_r$ and $a_m$ ($a_r^\dagger$ and $a_m^\dagger$) are, respectively, the creation and annihilation operators of the resonator (oscillator). The term $\hat{H}_{\text{coupling}}$ represents the oscillator-resonator coupling, and the term $\hat{H}_{\text{drive}}$ represents the driving force. We shall assume that the last two terms in the Hamiltonian are small: The smallness of $\hat{H}_{\text{coupling}}$ means that the energy eigenstates will, to a good approximation, be identified with well-defined excitation numbers in the oscillator and resonator, while the smallness of $\hat{H}_{\text{drive}}$ justifies a description of the system using time-independent energy levels. In the following we start by using thermodynamics arguments to derive an expression for the lower limit on the achievable temperature, and we later use a master-equation approach to treat the specific example discussed in Sec. II.

We first consider the situation depicted in Fig. 3. Each arrow describes a transition from a state $|i,j\rangle$ to another state $|i',j'\rangle$, where the meaning of the quantum numbers is explained in Fig. 3. We denote the rate at which such a transition occurs by $W_{|i,j\rangle \rightarrow |i',j'\rangle}$. In other words, the probability current of the transition is given by $P_{|i,j\rangle}W_{|i,j\rangle \rightarrow |i',j'\rangle}$. Where $P_{|i,j\rangle}$ is the occupation probability of the state $|i,j\rangle$. In the steady state, we can write detailed balance equations for the occupation probabilities of the different quantum states in the form

$$0 = \frac{dP_{|i,j\rangle}}{dt} = (W_{|i+1,j-1\rangle \rightarrow |i,j\rangle}P_{|i+1,j-1\rangle} - W_{|i,j\rangle \rightarrow |i+1,j-1\rangle}P_{|i,j\rangle})$$
$$+ (W_{|i-1,j+1\rangle \rightarrow |i,j\rangle}P_{|i-1,j+1\rangle} - W_{|i,j\rangle \rightarrow |i-1,j+1\rangle}P_{|i,j\rangle})$$
$$+ (W_{|j+1,i-1\rangle \rightarrow |j,i\rangle}P_{|j+1,i-1\rangle} - W_{|j,i\rangle \rightarrow |j+1,i-1\rangle}P_{|j,i\rangle}).$$  

Now we determine some relations among the rates $W$. Let us start with the situation when the driving force is switched off and the resonator is in contact with its surrounding environment, which is at temperature $T_r$. Assuming that the environment induces transitions between states that are different by one photon in the resonator as described in Fig. 3 and Eq. (16)], the rates must obey the thermal-equilibrium relation

$$\frac{W_{|i,j\rangle \rightarrow |i+1,j\rangle}}{W_{|i+1,j\rangle \rightarrow |i,j\rangle}} = \exp\left\{-\frac{\hbar \omega_r}{k_B T_r}\right\}.$$  

Note that these transitions do not change the state of the mechanical oscillator, since without the driving force the oscillator and resonator are effectively uncoupled ($\omega_m \ll \omega_r$). The oscillator is itself in contact with its environment at temperature $T_m$, but for optimal cooling we assume that the insulation is good enough that we can completely neglect environment-induced transitions, i.e., we have assumed that $W_{|i,j\rangle \rightarrow |i,j\rangle \rightarrow 0} \rightarrow 0$ in Fig. 3 and Eq. (16). We now assume that the driving force couples states of the form $|i,j\rangle$ and $|i+1,j-1\rangle$ but does not drive any other transitions (this assumption must be justified for a given model, as will be done...
below for the system of interest). Since the driving force is a classical one, the transitions it induces must have equal rates in both directions, i.e.,

\[ W_{|j, j+1\rangle \rightarrow |j+1, j\rangle} = W_{|j+1, j\rangle \rightarrow |j, j+1\rangle}. \]

(18)

The reason why there is no Boltzmann factor in Eq. (18) is that these transitions are mainly induced by the classical driving force, and any contributions to their rates from the thermal environment are negligible.

Using Eqs. (17) and (18), it is not difficult to verify that the pairs of terms in Eq. (16) all vanish when

\[ P_{|j, j\rangle} = \frac{1}{Z} \exp \left\{ -\frac{(i + j) \hbar \omega_r}{k_B T_r} \right\}, \]

(19)

where \( Z \) is the partition function. This steady-state probability distribution \( P_{|j, j\rangle} \) can now be rewritten as

\[ P_{|j, j\rangle} = \frac{1}{Z_r} \exp \left\{ -\frac{i \hbar \omega_r}{k_B T_r} \right\} \times \frac{1}{Z_m} \exp \left\{ -\frac{j \hbar \omega_m}{k_B T_m} \right\}, \]

(20)

with

\[ \frac{T_m}{T_r} = \frac{\omega_m}{\omega_r}. \]

(21)

Here \( Z_r \) and \( Z_m \) is partition the function of resonator and mechanical oscillator, respectively. We therefore find that if the above picture about the allowed transitions and the relations governing their rates are valid, we can reach the final temperature \( T_m \) given by Eq. (21).

The above derivation suggests an intuitive picture for the cooling mechanism. The purpose of the driving force is to facilitate the transfer of excitations between the resonator and oscillator. Before the driving starts, the low-frequency oscillator has many more excitations than the high-frequency resonator. Once the driving starts, the excitation imbalance causes excitations to start flowing from the oscillator to the resonator. As the number of excitations in the resonator goes above the thermal-equilibrium value, excitations start to dissipate from the resonator to the environment. A steady state is eventually reached with the resonator in thermal equilibrium with the environment and both the resonator and the oscillator having the same average number of excitations (in fact, the resonator and the oscillator will have the same excitation-number probability distribution). This picture of the cooling mechanism reveals another point that is generally not noted in the literature: Although \( T_r \) is desired to be smaller than \( \omega_m \) in order to avoid heating effects, it should not be too small, because it provides the mechanism by which excitations are dissipated from the resonator into the environment. In particular, it must be larger than \( T_m \), such that the dissipation of excitations is faster than the heating of the oscillator by its environment.

We now consider what would happen if one were able to drive the transitions shown in Fig. 4. With optimal parameters for cooling, one would obtain the minimum temperature
order to determine the transitions that can be driven by the external force, we need to evaluate matrix elements of the form \( \langle \psi_{i,j} | (a_i + a_i^\dagger) | \psi_{j,k} \rangle \), where we now use the eigenstates of the Hamiltonian (i.e., slightly modified from the case of two uncoupled oscillators). To first order in perturbation theory,

\[
\langle \psi_{i,j} | = | i,j \rangle + \frac{(2i+1)g}{\omega_m} (\sqrt{j|i,j-1|} - \sqrt{j+1|i,j+1|})
\]

\[
+ \frac{g}{2\omega_r - \omega_m} (\sqrt{(i-1)(j+1)|i-2,j+1|} - \sqrt{(i+1)(j+2)|i+2,j-1|}).
\]

Using the above approximation, we find that

\[
\langle \psi_{i,j} | a_i + a_i^\dagger | \psi_{j,k} \rangle = -\frac{2g \sqrt{(i+1)}}{\omega_m}. \tag{26}
\]

It is straightforward to see from Eq. (24) that

\[
\langle \psi_{i,j} | a_i + a_i^\dagger | \psi_{j,k} \rangle = 0.
\]

Using numerical calculations we find that

\[
\langle \psi_{i,j} | a_i + a_i^\dagger | \psi_{j,k} \rangle = -\frac{12g^3 (i+1)(i+2)(i+3)j}{(2\omega_r - \omega_m)^3}. \tag{27}
\]

The above results imply that the driving term can be used to drive transitions of the form \(| i,j \rangle \rightarrow | i+1,j-1 \rangle \), which can be used to remove excitations from the oscillator and add them to the resonator. These transitions correspond to the picture shown in Fig. 3, and their resonance frequency is given by \( \omega_r = \omega_m - \omega_m \). The steady-state effective temperature for the oscillator is given by Eq. (21) when driving these transitions, assuming that heating effects are avoided. By driving the system at the frequency \( \omega_r = 3\omega_m - \omega_m \), one could in principle drive the transitions \(| i,j \rangle \rightarrow | i+3,j-1 \rangle \) and reach a lower minimum temperature. However, the fact that the corresponding matrix element is proportional to the third power of the small coupling strength \( g \) suggests that this matrix element will be extremely small for any realistic parameters, hindering the possibility of utilizing this cooling mechanism.

We now turn to the heating effects that have been neglected above. We note that the driving term in Eq. (24) can also drive transitions of the form \(| i,j \rangle \rightarrow | i+1,j+1 \rangle \), and the relevant matrix element is given by

\[
\langle \psi_{i,j} | a_i + a_i^\dagger | \psi_{j,k} \rangle = \frac{2g \sqrt{(i+1)(j+1)}}{\omega_m}. \tag{28}
\]

These undesired transitions are induced if either the driving amplitude \( A \) or the resonator’s damping rate \( \Gamma_r \) is comparable to or larger than \( \omega_m \). If either one or both of the above conditions are satisfied, the driving force must be considered within the resonance region of the above transition. As a result, additional excitations would be steadily pumped into the system, resulting in a higher temperature than what would be obtained from the simple picture of transition rates that we have presented above (note that this heating is a consequence of the nonlinearity in the system Hamiltonian). \(^{29}\) The ideal parameters for cooling are therefore given by \( \omega_p = \omega_m - \omega_m \), \( \Delta \ll (\omega_m - \omega_p) \), and \( \Gamma_r \ll (\omega_m - \omega_p) \); naturally, \( \Gamma_m \) is desired to be much smaller than the smallest of three parameters that determine the cooling power: \( \Gamma_r, \Delta g/\omega_m \), and \((\Delta g)^2/(\Gamma_m \omega_m^2)\). This condition on \( \Gamma_m \) ensures that the oscillator heating from its contact with the environment is slower than the cooling it experiences as a result of the driving. Using a numerical simulation, we shall see shortly that the above heating effects can be made negligible with the proper choice of parameters. We should also mention here that in this section we have not considered the noise in the driving force, i.e., we have assumed an ideal microwave source. Such noise would directly heat the resonator, resulting in a higher base temperature, as discussed in Sec. II. It is also important to note that the linearity of the resonator, i.e., the cooler, helps reduce its direct heating by the driving force. If one uses a nonlinear system, e.g., a Cooper-pair box, this direct heating mechanism (even if an ideal microwave signal is used) could be the most dominant heating mechanism. \(^{29,30}\)

In order to give a concrete example that illustrates the cooling dynamics, we now turn to a master-equation approach. The density matrix \( \rho \) of the system evolves in time according to the master equation

\[
\frac{d\rho}{dt} = -\frac{i}{\hbar} [\hat{H},\rho] + (1 + \bar{N}_r) \Gamma_r \left( a_r^\dagger \rho a_r - \frac{1}{2} a_r^\dagger a_r \rho - \frac{1}{2} \rho a_r a_r^\dagger \right)
\]

\[
+ \bar{N}_r \Gamma_r \left( a_r^\dagger \rho a_r - \frac{1}{2} a_r^\dagger a_r \rho - \frac{1}{2} \rho a_r a_r^\dagger \right)
\]

\[
+ (1 + \bar{N}_m) \Gamma_m \left( a_m^\dagger \rho a_m - \frac{1}{2} a_m^\dagger a_m \rho - \frac{1}{2} \rho a_m a_m^\dagger \right)
\]

\[
+ \bar{N}_m \Gamma_m \left( a_m^\dagger \rho a_m - \frac{1}{2} a_m^\dagger a_m \rho - \frac{1}{2} \rho a_m a_m^\dagger \right), \tag{29}
\]

where

\[
\bar{N}_r = \frac{1}{e^{\hbar \nu_r / kT} - 1} \tag{30}
\]

and similarly for \( \bar{N}_m \). The coefficients \( \Gamma_r \) and \( \Gamma_m \) are decay rates for the resonator and oscillator, respectively.

An example illustrating the dynamics of cooling the mechanical oscillator by the microwave resonator is shown in Fig. 5. The results were obtained by numerically solving Eq. (29) using the Hamiltonian in Eq. (24). The effective temperatures of the oscillator and resonator are obtained by calculating their respective entropies from their reduced density matrices \( S = -\text{Trace}(\rho \log \rho) \) and fitting these values to the temperature-entropy relation for a harmonic oscillator. The initial heating of the resonator is a result of the transfer of excitations from the oscillator to the resonator. For large \( t \), the system reaches a steady state where the ratio between the effective temperatures of the oscillator and the resonator is approximately equal to \( \omega_m / \omega_r \).
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is limited by the ratio \( \omega_p / \omega_0 \). This lower limit for the cooling becomes crucial for radio frequency and microwave resonators pumped by a real (noisy) microwave source since their effective temperature \( T_r \) is usually much larger than the ambient temperature. We should also emphasize that our results apply, with minor modifications, to other types of coolers, e.g., a Cooper-pair box.

Note added in proof. Recently some related manuscripts appeared on the e-print archive 31–34

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FIG. 5. Effective temperatures for the electromagnetic resonator (solid line) and the mechanical oscillator (dashed line), relative to the ambient temperature, as functions of time \( t \) for the parameters \( \omega_m / \omega_0 = 0.1, g / \omega_0 = 0.005, \Gamma_m / \omega_0 = 0.5, \Gamma_r / (\omega_0 / 2\pi) = 0.05 \), and \( \Gamma_r = 0 \). The driving field, with amplitude \( A / \omega_0 = 0.05 \) and frequency \( \omega_m = \omega_0 - \omega_m \), is turned on at \( t = 0 \).

IV. CONCLUSIONS

We have shown that both the classical and the quantum treatment give the same final result: the cooling factor \( T_m / T_r \)
In this paper we use the convention where linear coupling between two harmonic oscillators means a coupling term of the form $g(a_0 + a_0^\dagger)(\beta a_0 + \beta^\dagger a_0^\dagger)$ and linear coupling to the driving force means a term of the form $(\xi a_0 + \xi^\dagger a_0^\dagger + \eta a_0 + \eta^\dagger a_0^\dagger)\cos(\omega_0 t + \theta)$, where $\alpha$, $\beta$, $\xi$, $\eta$, $\omega_0$, and $\theta$ are constants. Under this convention parametric driving, i.e., when the coupling strength $g$ in Eq. (24) is modulated by an externally applied signal, corresponds to a nonlinear driving term. Although such parametric driving can be used to implement the sideband cooling mechanism, in this paper we are only considering the case where the externally applied driving signal couples to the charge on the resonator, i.e., a linear driving term according to the convention that we follow. Since we further assume a linear mechanical oscillator and linear coupling to the environment (in the case of linear oscillators), a nonlinearity is needed either in the coolant Hamiltonian or the coupling term.